Elastic waves in solids Crash Course

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Outline of the Lecture

I. Elastic Wave Propagation in Free Space

- 1. 3D Elasticity
- 2. Longitudinal and Transverse Waves

II. Reflection and transmission through interfaces

- 1. Boundary Conditions
- 2. Free Surface Reflection
- 3. Solid-Fluid Interface

III.Guided Waves

- 1. Surface Waves
- 2. Lamb Waves

Elastic Waves in Solids: 1D Solid

$$\sigma(x) \qquad \sigma(x+dx)$$

$$x \qquad x+dx$$

1D medium along x, independent of y or z.

$$dm \frac{\partial^2 u}{\partial t^2} = \sigma(x+dx)A - \sigma(x)A \quad dm = \rho A \, dx$$

$$\rho A \frac{\partial^2 u}{\partial t^2} = \frac{\sigma(x+dx) - \sigma(x)}{dx}A = \frac{\partial \sigma}{\partial x}A$$

$$\rho \frac{\partial^2 u}{\partial t^2} = \frac{\partial \sigma}{\partial x} = \frac{\partial(E\varepsilon)}{\partial x} = E \frac{\partial}{\partial x} (\frac{\partial u}{\partial x})$$

$$\frac{\partial^2 u}{\partial t^2} = \frac{\partial \sigma}{\partial x} = \frac{\partial(E\varepsilon)}{\partial x} = E \frac{\partial}{\partial x} (\frac{\partial u}{\partial x})$$

$$\frac{\partial^2 u}{\partial t^2} = \frac{E}{\rho} \frac{\partial^2 u}{\partial x^2}$$

3D solid medium

 $\vec{u}(\vec{x})$: displacement from equilibrium at position x. $\vec{x} = (x_1, x_2, x_3)$

 $\vec{u}(\vec{x},t) = (u_1(\vec{x},t), u_2(\vec{x},t), u_3(\vec{x},t))$

$$u_i(\vec{x} + d\vec{x}) = u_i(\vec{x}) + \frac{\partial u_i}{\partial x_j} dx_j$$

No deformation if the gradient of displacement is zero.

$$du_{i}(\vec{x}+d\vec{x}) = \frac{\partial u_{i}}{\partial x_{j}} dx_{j} = \frac{1}{2} \left(\frac{\partial u_{i}}{\partial x_{j}} + \frac{\partial u_{j}}{\partial x_{i}} \right) dx_{j} + \frac{1}{2} \left(\frac{\partial u_{i}}{\partial x_{j}} - \frac{\partial u_{j}}{\partial x_{i}} \right) dx_{j}$$

$$Symmetric$$

$$Antisymmetric$$

$$Strain tensor \quad \varepsilon_{ij} = \frac{1}{2} \left(\frac{\partial u_{i}}{\partial x_{j}} + \frac{\partial u_{j}}{\partial x_{i}} \right)$$

$$\omega_{ij} = \frac{1}{2} \left(\frac{\partial u_{i}}{\partial x_{j}} - \frac{\partial u_{j}}{\partial x_{i}} \right)$$

$$Describe a rotation movement$$

$$\rightarrow \text{ no vibrations}$$

Stress Tensor

 $\vec{dF} = \vec{n} \, \underline{\sigma} \, dS$



Elastic wave equation

 $\vec{dF} = \vec{n} \, \underline{\sigma} \, dS$



The total stress force acting on the body of volume V is: $\iint_{S} \vec{n} \underline{\sigma} \, dS = \iiint_{V} \operatorname{div} \underline{\sigma} \, dV$ Gauss theorem Balance of forces: $\iiint_{V} \rho \frac{\partial^{2} \vec{u}}{\partial t^{2}} \, dV = \iiint_{V} \operatorname{div} \underline{\sigma} \, dV + \iiint_{V} \vec{f} \, dV$

with f is the density of external forces

$$\rho \frac{\partial^2 \vec{u}}{\partial t^2} = \operatorname{div} \underline{\sigma} + \vec{f} \qquad \rho \frac{\partial^2 u_i}{\partial t^2} = \frac{\partial \sigma_{ij}}{\partial x_i} + f_i$$

Symmetry of the stress tensor

When there is no density of moment, we can show that the stress tensor is symmetric:

$$\sigma_{ij} = \sigma_{ji}$$
 i, j=1,2,3

Only 6 independent components:

- 3 Normal stresses $\sigma_{11}, \sigma_{22}, \sigma_{33}$
- 3 Tangential stresses $\sigma_{12} = \sigma_{21}, \sigma_{13} = \sigma_{31}, \sigma_{23} = \sigma_{32}$

Linear Elasticity

Assuming a linear link between stress tensor and deformation

$$\sigma_{ij} = c_{ijkl} \varepsilon_{kl}$$

 C_{ijkl} : Elasticity tensor, fourth-rank (3⁴ = 81 terms)

From the definition, we have the same symmetries as σ_{ii} and

$$\mathcal{E}_{kl}$$
: $C_{ijkl} = C_{jikl} = C_{ijlk}$

By considering a reversible transformation, we can also show:

$$C_{ijkl} = C_{klij}$$
81 terms \rightarrow 21 terms
$$\sigma_{ij} = c_{ijkl} \varepsilon_{kl} = \frac{1}{2} c_{ijkl} \left(\frac{\partial u_k}{\partial x_l} + \frac{\partial u_l}{\partial x_k} \right) = c_{ijkl} \frac{\partial u_k}{\partial x_l}$$

Elasticity Notations

Voigt notation using the fundamental symmetries

of σ and \mathcal{E} :

$$\begin{pmatrix} \sigma_{11} \\ \sigma_{22} \\ \sigma_{33} \\ \sigma_{23} \\ \sigma_{31} \\ \sigma_{12} \end{pmatrix} = \begin{pmatrix} c_{11} & c_{12} & c_{13} & c_{14} & c_{15} & c_{16} \\ c_{21} & c_{22} & c_{23} & c_{24} & c_{25} & c_{26} \\ c_{31} & c_{32} & c_{33} & c_{34} & c_{35} & c_{36} \\ c_{41} & c_{42} & c_{43} & c_{44} & c_{45} & c_{46} \\ c_{51} & c_{52} & c_{53} & c_{54} & c_{55} & c_{56} \\ c_{61} & c_{62} & c_{63} & c_{64} & c_{65} & c_{66} \end{pmatrix} \begin{pmatrix} \varepsilon_{11} \\ \varepsilon_{22} \\ \varepsilon_{33} \\ \varepsilon_{23} \\ \varepsilon_{31} \\ \varepsilon_{12} \end{pmatrix}$$

Most general notation, for any kind of material

- Isotropic: 2 coefficients
- Axi-symetrical: 5 coefficients (Glass wool)
- Orthotropic: 9 coefficients (Wood)

For an isotropic medium, the elasticity tensor can be written as:

$$c_{ijkl} = \lambda \,\delta_{ij} \,\delta_{kl} + \mu \left(\delta_{ik} \,\delta_{jl} + \delta_{il} \,\delta_{jk} \right)$$

Related to compression

with the Lamé coefficients λ and μ — Related to shear

$$\sigma_{ij} = c_{ijkl} \varepsilon_{kl} = \lambda \,\delta_{ij} \,\delta_{kl} \varepsilon_{kl} + \mu (\delta_{ik} \,\delta_{jl} + \delta_{il} \,\delta_{jk}) \varepsilon_{kl}$$

$$\sigma_{ij} = \lambda \varepsilon_{kk} + 2 \mu \varepsilon_{ij}$$
 or $\underline{\sigma} = \lambda \operatorname{Id} \operatorname{Tr} \underline{\varepsilon} + 2 \mu \underline{\varepsilon}$

In terms of u, we can write:

$$\vec{\operatorname{div}} \,\underline{\sigma} = (\lambda + \mu) \vec{\operatorname{grad}} (\operatorname{div} \vec{u}) + \mu \Delta \vec{u}$$

Hooke's Law



Suitable for isotropic materials (2 parameters only)

Possibility to link the different notations, e.g.:

$$\lambda = \frac{\nu E}{(1 - 2\nu)(1 + \nu)} = c_{11} - 2c_{44}, \qquad \mu = \frac{E}{2(1 + \nu)} = c_{44}$$

Conversion formula

Uncoupling of the Wave Equation

Wave equation:

$$\rho \frac{\partial^2 \vec{u}}{\partial t^2} = \vec{\operatorname{div}} \,\underline{\sigma} + \vec{f}$$
$$\rho \frac{\partial^2 \vec{u}}{\partial t^2} = (\lambda + \mu) \vec{\operatorname{grad}} (\operatorname{div} \vec{u}) + \mu \vec{\Delta} \vec{u} + \vec{f}$$

Laplacian can be written as: $\vec{\Delta} \vec{u} = \text{grad}(\text{div} \vec{A}) - \text{rot}(\vec{rot} \vec{A})$

$$\rho \frac{\partial^2 \vec{u}}{\partial t^2} = (\lambda + 2\mu) \vec{\text{grad}} (\text{div} \, \vec{u}) - \mu \, \vec{\text{rot}} (\vec{\text{rot}} \, \vec{u}) + \vec{f}$$

Helmholtz Decomposition

We write the displacement field as the sum of two terms:

$$\vec{u} = \vec{u_L} + \vec{u_T} \qquad \vec{u_L} = \vec{\text{grad}} \phi$$
$$\vec{u_T} = \vec{\text{rot}} \vec{\psi}$$

 ϕ is a scalar potential, ψ a vector potential.

$$\operatorname{rot} \vec{u_L} = \vec{0}, \quad \operatorname{div} \vec{u_T} = 0$$

Thus, we can separate the two components:

$$\begin{cases} \frac{\partial^2 \vec{u_L}}{\partial t^2} - c_L \operatorname{grad}(\operatorname{div} \vec{u_L}) = 0, \quad c_L = \sqrt{\frac{\lambda + 2\mu}{\rho}} \\ \frac{\partial^2 \vec{u_L}}{\partial t^2} + c_T^2 \operatorname{rot}(\operatorname{rot} \vec{u_T}) = 0, \quad c_T = \sqrt{\frac{\mu}{\rho}} \end{cases}$$

Helmholtz Decomposition

We obtain d'Alembert equation for displacements and potential:

$$\begin{cases} \frac{\partial^2 \vec{u}_L}{\partial t^2} - c_L \vec{\Delta} \vec{u}_L = \vec{0}, & \frac{\partial^2 \phi}{\partial t^2} - c_L \Delta \phi = 0\\ \frac{\partial^2 \vec{u}_T}{\partial t^2} - c_T^2 \vec{\Delta} \vec{u}_T = \vec{0}, & \frac{\partial^2 \vec{\psi}}{\partial t^2} - c_T^2 \vec{\Delta} \vec{\psi} = \vec{0} \end{cases}$$

We replaced 3 unknown displacement components by 4 potential components (1 for scalar, 3 for vectorial). A gauge equation might be needed:

div
$$\vec{\psi} = 0$$

Examples of Material Properties

Material	ρ (g/cm ³)	λ(GPa)	μ(GPa)	<i>c_L</i> (mm/µs)	<i>c_T</i> (mm/µs)
Steel	7.5-9	95-140	75-90	5.2-6.5	2.9-3.5
Aluminium	2.7	58	26	6.3	3.1
Concrete	1.3-2.5	5-14	8-21	2.9-6.5	1.8-4
Diamond	3.5	340	510	19.7	12.1
Ероху	1.1-1.4	1.4-3.6	0.7-1.9	1.4-2.6	0.7-1.3
Plexiglass	1.2	3.4-7.8	0.8-1.2	2-2.9	0.8-1
Lead	11.5	39	6.3	2.1	0.75
PVC	1.1-1.5	1.7-5.7	0.7-1.5	1.4-2.8	0.7-1.2
Titanium	4.5	77	43	6	3
Glass	2.4-2.8	14-25	20-38	4.4-6.5	2.7-4

As for acoustic waves, plane wave are solutions for infinite space:

$$\vec{u}(\vec{x},t) = A \vec{P} e^{(\vec{k}\vec{x}-\omega t)}$$

A is the amplitude, \vec{P} a polarisation vector depending on \vec{n} as $\vec{k} = k \vec{n}$.

For longitudinal waves:
$$\vec{rot} \vec{u}_L = \vec{0} \implies \vec{n} \land \vec{P}_L = \vec{0}$$

For transverse waves: $\operatorname{div} \vec{u_T} = 0 \implies \vec{n} \cdot \vec{P_T} = 0$

Animations

Example:
$$\vec{n} = \vec{e_1} \Rightarrow \vec{P_L} = \vec{e_1}, \vec{P_{T_1}} = \vec{e_2} \text{ and } \vec{P_{T_2}} = \vec{e_3}$$

Transverse Waves



https://www.acs.psu.edu/drussell/demos.html





Longitudinal Waves



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Boundary Conditions

Let's consider a surface S between two solids, normal \vec{n} . The continuity conditions for any points in S write:

$$\begin{cases} \underline{\sigma_1}(\vec{x},t)\vec{n} = \underline{\sigma_2}(\vec{x},t)\vec{n} \\ \vec{u_1}(\vec{x},t) = \vec{u_2}(\vec{x},t) \end{cases}$$



$$\begin{cases} \underline{\sigma_1}(\vec{x},t)\vec{n} = -P_2(\vec{x},t)\vec{n} \\ \vec{u_1}(\vec{x},t).\vec{n} = \vec{u_2}(\vec{x},t).\vec{n} \end{cases}$$

No shear stress in a fluid, and only normal displacement continuity.

2

 \vec{n}

Free-surface reflection

Reflection on a free space: $\underline{\sigma}(\vec{x},t).\vec{e_3}=\vec{0}$

ncident plane wave:
$$\vec{u}_{I}(\vec{x},t) = A_{I}\vec{P}_{I}e^{(k\vec{n}_{I}\vec{x}-\omega t)}$$

$$\vec{P} = \begin{pmatrix} P_1 \\ P_2 \\ P_3 \end{pmatrix}$$
, with $\vec{n}_I = \begin{pmatrix} \sin \theta_I \\ 0 \\ \cos \theta_I \end{pmatrix}$, and $k_I = \omega/c_I$



For each point in S,

Snell-Descartes Law

$$\vec{k}_I \cdot \vec{x} = \vec{k}_L \cdot \vec{x} = \vec{k}_T \cdot \vec{x}$$
, then $\frac{\sin \theta_I}{c_I} = \frac{\sin \theta_L}{c_L} = \frac{\sin \theta_T}{c_T}$

Free-surface reflection

Reflected waves:

$$\vec{u}_{R}(x_{1}, x_{3}, t) = \vec{u}_{L}(x_{1}, x_{3}, t) + \vec{u}_{T_{1}}(x_{1}, x_{3}, t) + \vec{u}_{T_{2}}(x_{1}, x_{3}, t)$$

$$\begin{cases} \vec{u}_{L}(\vec{x}, t) = A_{L}\vec{P}_{L}e^{(\vec{k}_{1}.\vec{x}-\omega t)} \\ \vec{u}_{T_{1}}(\vec{x}, t) = A_{T_{1}}\vec{P}_{T_{1}}e^{(\vec{k}_{1}.\vec{x}-\omega t)} \\ \vec{u}_{T_{2}}(\vec{x}, t) = A_{T_{2}}\vec{P}_{T_{2}}e^{(\vec{k}_{1}.\vec{x}-\omega t)} \end{cases}$$

Using:
$$\vec{k}_{L} \wedge \vec{P}_{L} = \vec{0}$$
 P
 $\vec{k}_{T} \cdot \vec{P}_{T} = 0$ $\vec{P}_{L} = \frac{C_{L}}{\omega} \begin{pmatrix} k_{1} \\ 0 \\ -k_{3L} \end{pmatrix}$, $\vec{P}_{T_{1}} = \frac{C_{T}}{\omega} \begin{pmatrix} k_{3T} \\ 0 \\ -k_{1} \end{pmatrix}$, $\vec{P}_{T_{2}} = \begin{pmatrix} 0 \\ 1 \\ 0 \end{pmatrix}$

 $\mathbf{r} X_1$

Ħ

Free-surface reflection



Incident Longitudinal Wave



Incident Longitudinal Wave



- The amplitude is maximal for normal incidence or grazing incidence (no reflected shear wave).
- The conversion to shear waves can be perfect for specific angles as $r_{LL}=0$, if $\frac{c_L}{c_T}>0.565$

Incident Transverse Vertical Wave

$$\sin\theta_L = \frac{c_L}{c_T} \sin\theta_T$$

As shear waves are usually slower than compressive waves, $\theta_L > \theta_T$.

A critical incident angle is reached as
$$\theta_L = 90^\circ$$
,
 $\theta_T^c = \arcsin\left(\frac{c_T}{c_L}\right)$

Then reflected compressive waves becomes evanescent.



Solid-Liquid, Liquid-Solid interfaces



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Specific shapes are waveguides, based on the coupling between propagation properties and boundary conditions

Two free surfaces guide the wave through successive reflections

One free surface is already a waveguide !



$$\rho \frac{\partial^{2} \vec{u}}{\partial t^{2}} = (\lambda + \mu) \operatorname{grad} (\operatorname{div} \vec{u}) + \mu \vec{\Delta} \vec{u}$$
$$\vec{u} = \operatorname{grad} \phi_{L} + \operatorname{rot} \vec{\psi}$$
$$u_{x} = \frac{\partial \phi_{L}}{\partial x} + \frac{\partial \psi_{z}}{\partial y} - \frac{\partial \psi_{y}}{\partial z} = \frac{\partial \phi_{L}}{\partial x} - \frac{\partial \psi_{y}}{\partial z}$$
$$u_{y} = \frac{\partial \phi_{L}}{\partial y} + \frac{\partial \psi_{x}}{\partial z} - \frac{\partial \psi_{z}}{\partial x} = \frac{\partial \psi_{x}}{\partial z} - \frac{\partial \psi_{z}}{\partial x}$$
$$u_{z} = \frac{\partial \phi_{L}}{\partial z} + \frac{\partial \psi_{y}}{\partial x} - \frac{\partial \psi_{z}}{\partial y} = \frac{\partial \phi_{L}}{\partial z} + \frac{\partial \psi_{y}}{\partial x}$$



Polarization in (x, z) plane $\Rightarrow u_y = 0$ We can choose

$$\vec{\Psi} = \phi_T \vec{e}_y$$

$$\Delta \phi - \frac{1}{c^2} \frac{\partial^2 \phi}{\partial t^2} = 0 \quad \Rightarrow$$

$$\frac{\partial^2 \phi_M}{\partial x^2} + \frac{\partial^2 \phi_M}{\partial z^2} - \frac{1}{c_M^2} \frac{\partial^2 \phi_M}{\partial t^2} = 0 \quad \text{with} \quad L = M, T$$

We look for solutions in the form:

$$\phi_M(x,z,t) = f_M(z)e^{i(kx-\omega t)}$$

Propagative harmonic waves toward x.

$$\frac{\partial^2 f_M}{\partial z^2} + \left(\frac{\omega^2}{c_M^2} - k^2\right) f_M(z) = 0 \quad \text{with} \quad L = M, T$$

$$f_M(z) = A_M e^{-\alpha_M z} + B_M e^{\alpha_M z}$$
 with $\alpha_M = \sqrt{k^2 - \frac{\omega^2}{c_M^2}} = k \sqrt{1 - \frac{c^2}{c_M^2}}$

B_M must be zero to avoid divergence.

We can write: $\phi_M(x, z, t) = A_M e^{-\alpha_M z} e^{i(kx - \omega t)}$

$$\vec{u}(x,z,t) = \left[\begin{pmatrix} ik \\ 0 \\ -\alpha_L \end{pmatrix} A_L e^{-\alpha_L z} + \begin{pmatrix} \alpha_T \\ 0 \\ ik \end{pmatrix} A_T e^{\alpha_T z} \right] e^{i(kx - \omega t)}$$

Boundary conditions: $\underline{\sigma}(\vec{x},t) \cdot \vec{e}_z = \vec{0}$,

Elasticity law gives:

$$\begin{cases} \sigma_{xz} = \mu \left(\frac{\partial u_z}{\partial x} + \frac{\partial u_x}{\partial z} \right) & \text{y invariance and no} \\ \sigma_{yz} = \mu \left(\frac{\partial u_z}{\partial y} + \frac{\partial u_y}{\partial z} \right) & \text{displacement in y direction} \\ \sigma_{zz} = \lambda \frac{\partial u_x}{\partial x} + (\lambda + 2\mu) \frac{\partial u_z}{\partial z} \end{cases}$$

Injecting the waveform:

$$\begin{cases} \sigma_{xx}(x, z, t) = \mu [-2ik \alpha_L A_L e^{-\alpha_L z} - (k^2 + \alpha_T^2) A_T e^{-\alpha_L z}] e^{i(kx - \omega t)} \\ \sigma_{zz}(x, z, t) = \mu [(k^2 + \alpha_T^2) \alpha_L A_L e^{-\alpha_L z} - 2ik A_T \alpha_T e^{-\alpha_L z}] e^{i(kx - \omega t)} \end{cases}$$

At z=0,
$$\begin{cases} 2ik\alpha_L A_L + (k^2 + \alpha_T^2)A_T = 0\\ (k^2 + \alpha_T^2)\alpha_L A_L - 2ik\alpha_T A_T = 0 \end{cases}$$

Solving the system provides the dispersion equation:

$$(k^2 + \alpha_T^2)^2 - 4k^2 \alpha_L \alpha_T = 0$$

In terms of wave velocity:

$$\left(2 - \frac{c^2}{c_T^2}\right)^2 - 4\sqrt{\left(1 - \frac{c^2}{c_L^2}\right)\left(1 - \frac{c^2}{c_T^2}\right)} = 0$$

Only one solution c_R , depending on c_L and c_T , independent of the frequency

Approximate solution:

$$\frac{c_R}{c_T} = \sqrt{\frac{1.44 \,\lambda + 0.88 \,\mu}{1.58 \,\lambda + 1.16 \,\mu}}$$

$$\begin{pmatrix} U_x(x,z,t) = -(ke^{-\alpha_L z} - \sqrt{\alpha_L \alpha_T}e^{-\alpha_T z})A_L \sin(kx - \omega t) \\ U_z(x,z,t) = \left(-\alpha_L e^{-\alpha_L z} + k\sqrt{\frac{\alpha_L}{\alpha_T}}e^{-\alpha_T z}\right)A_L \cos(kx - \omega t) \end{cases}$$

- Evanescent part in z direction
- Phase quadrature between x and z: elliptic polarization
- Possible sign changes

 $\begin{cases} U_x(x,z,t) = -(ke^{-\alpha_L z} - \sqrt{\alpha_L \alpha_T}e^{-\alpha_T z})A_L \sin(kx - \omega t) \\ U_z(x,z,t) = \left(-\alpha_L e^{-\alpha_L z} + k\sqrt{\frac{\alpha_L}{\alpha_T}}e^{-\alpha_T z}\right)A_L \cos(kx - \omega t) \end{cases}$

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Scholte and Stoneley Waves



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Leaky Waves

Surface guided wave, decreasing with z and x



SH Guided Waves



From Noé Jiménez website https://nojigon.webs.upv.es/ Cut-off at low frequency for m > 0



Deriving the displacement from the potentials:

$$\begin{aligned} u_{x} &= \frac{\partial \phi_{L}}{\partial x} - \frac{\partial \phi_{T}}{\partial z} = i k \phi_{L} - \frac{\partial \phi_{T}}{\partial z} \\ u_{z} &= \frac{\partial \phi_{L}}{\partial z} + \frac{\partial \phi_{T}}{\partial x} = \frac{\partial \phi_{L}}{\partial z} + i k \phi_{T} \end{aligned}$$

We only need these two components of the stress to express boundary conditions:

$$\sigma_{xz} = \mu \left(\frac{\partial u_z}{\partial x} + \frac{\partial u_x}{\partial z} \right) = \mu \left((q^2 - k^2) \phi_T + 2ik \frac{\partial \phi_L}{\partial z} \right)$$

$$\sigma_{zz} = \lambda \frac{\partial u_x}{\partial x} - (\lambda + 2\mu) \frac{\partial u_z}{\partial z} = \mu \left((k^2 - q^2) \phi_L + 2ik \frac{\partial \phi_T}{\partial z} \right)$$

Boundary conditions:

$$\begin{cases} \sigma_{xz}(z=\pm h/2)=0\\ \sigma_{zz}(z=\pm h/2)=0 \end{cases}$$

Can be verified only if the two potentials have different parities:

$$f_{L}(z) = B\cos(pz+\alpha) \text{ and } f_{T}(z) = A\sin(qz+\alpha) \text{ with } \alpha = 0 \text{ or } \pi/2$$

$$\begin{cases}
u_{x} = \frac{\partial \phi_{L}}{\partial x} - \frac{\partial \phi_{T}}{\partial z} = (ikB\cos(pz+\alpha) - qA\cos(qz+\alpha))e^{i(kx-\omega t)} \\
u_{z} = \frac{\partial \phi_{L}}{\partial z} + \frac{\partial \phi_{T}}{\partial x} = (pB\sin(pz+\alpha) + ikA\sin(qz+\alpha))e^{i(kx-\omega t)}
\end{cases}$$

Boundary conditions:

$$\begin{cases} \sigma_{xz}(z=\pm h/2)=0\\ \sigma_{zz}(z=\pm h/2)=0\\ \begin{cases} (q^2-k^2)A\sin(qh/2+\alpha)-2ikBp\sin(ph/2+\alpha)=0 \end{cases} \end{cases}$$

 $\left[\left(k^2 - q^2 \right) B \cos\left(p h/2 + \alpha \right) + 2ik A q \cos\left(q h/2 + \alpha \right) = 0 \right]$

Solving the system gives the dispersion relation of both modes:

$$\frac{\omega^4}{c_T^4} = 4k^2 q^2 \left(1 - \frac{p}{q} \frac{\tan(ph/2 + \alpha)}{\tan(qh/2 + \alpha)} \right) \text{ with } \alpha = 0 \text{ or } \pi/2$$



Anti-symmetrical modes



- No crossings for each type of mode
- Cut-off frequency for all but the first modes A₀ and S₀
- Growing with k most of the time



- Daniel Royer, & Eugène Dieulesaint, Ondes élastiques dans les solides, Tome 1, Propagation libre et guidée.pdf.
- Notes from Master SPI, Tony Valier-Brasier, Sorbonne Université, available online
- Daniel Royer, Tony Valier-Brasier, Elastic Waves in Solids 1: Propagation
- Géradin, M. et Rixen, D. J. (2014). Mechanical vibrations : theory and application to structural dynamics. John Wiley & Sons.